Munich University of Applied Sciences

Faculty 09, Degree Programme WI, LM, AU Exam. Engineering Mechanics, SS 2010

Prof. Dr. Anzinger, Hoffmann, Simon

Date: 08.07.2010

Time to finish: 90 min., Resources: pocket calculator, formulary

First Name: Room-Nr.: Room

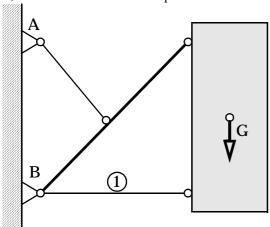
The valid Student ID together with a photo ID have to be presented during the examination!

Signature: Supervisor:

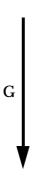
Task 1: Graphic Solution (12 points)

A body with the dead weight G=4kN is fixed in the depicted structure.

- a) Determine the support reactions in the bearing A and B graphically!
- b) Determine the force S_1 in strut 1!



Plan of forces: scale 1 cm = 1 kN



Results: $A = \overset{\mathcal{F}.8 \text{ kN}}{\dots}$; $B = \overset{5.4 \text{ kN}}{\dots}$; $S_1 = \overset{1.1 \text{ kN}}{\dots}$

Task 2: Rolling resistance (12 points)

A worker moves a wheelbarrow with constant speed.

Given:

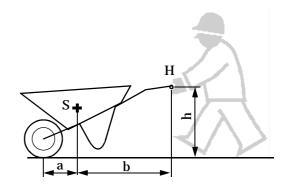
Weight of the wheelbarrow

(in centroid S) G = 1000 N,

Coefficient of rolling resistance $\mu_R = 0.07$, Distances: $a = 400 \,\text{mm}$; $b = 1000 \,\text{mm}$; $h = 830 \,\text{mm}$.

To be determined: Resultant of handforce H.

[H= 260 N]



Task 3: Shear pin (7 points)

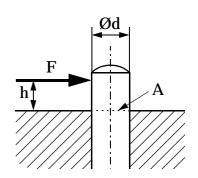
A cylindric steel pin is clamped rigidly and subject to a force F as shown.

Given: $\emptyset d = 10 \text{mm}$; h = 4 mm; F = 8 kN; flexural yield strength: $s_{bF} = 400 \text{ N/mm}^2$.

To be determined for the cross section A at the rigid clamping:

- a) Safety factor S_{Fb} against plastic flow due to max. bending stress S_b ,
- b) Safety factor S_{Fq} against plastic flow due to max. shear $\mbox{ stress }t_{q}$.

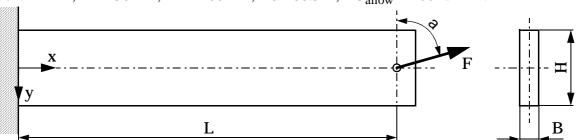
[a)
$$S_{Fb} = 1.23$$
 b) $S_{Fq} = 1.70$]



Task 4: Combined normal stress (16 points)

The sketched cantilever is subject to a force F as shown.

Given: L=1m; B=50mm; H=200mm; $a=80.54^{\circ}$; $s_{allow} = 100 \text{N/mm}^2$.



- a) Sketch in into the drawing at any position x: the distribution of the total normal stress over the height of the beam qualitatively.
- b) Calculate the location x0 at which compressive stress begins to occur.

$$[x_0 = 800 \text{ mm}]$$

c) Calculate the maximum of the force F so that the allowable total normal stress s_{allow} is not exceeded.

$$[F = 169 \text{ kN}]$$

Task 5: Stress resultants (25 points)

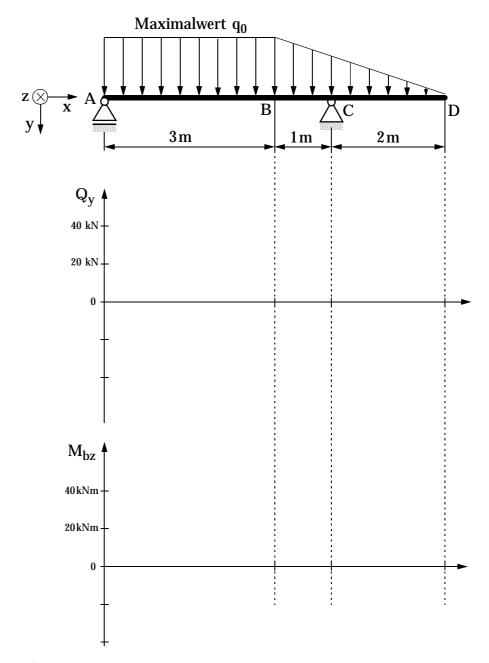
Given: A girder with both a constant and a linear decreasing line load q(x);

 $q_0 = 24 \text{kN/m}$; Distances as shown.

To be determined: a) the support reactions in the bearings A and C,

b) Profile of both transverse force Q_y and bending moment M_{bz} including the numerical values in B and C

c) exact position and numerical value of the maximum bending moment M_{bz}.



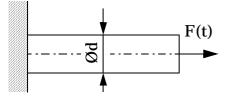
[a) A=45kN C=63kN c) $M_{0zmax}=42.2$ kNm at x=1.875m]

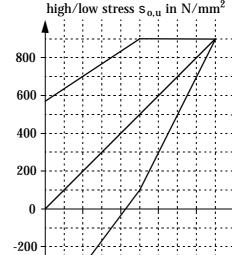
Task 6: Long life fatigue strength (11 points)

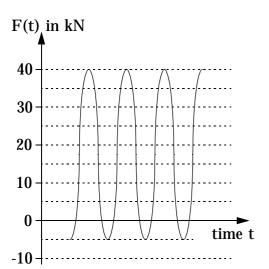
A girder (diameter d=10.55 mm) is subject to a dynamic compressive-/tensile-force F(t) (with constant mean load). Furthermore the Smith-diagram of the used material is given.

-400

-600







a) Calculate the safety factor S_D against endurance failure.

$$[S_D = 1.94]$$

b) Calculate the safety factor S_F against plastic flow.

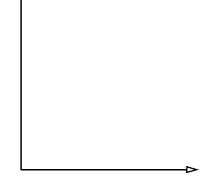
$$[S_F = 1,.97]$$

Task 7: Tensile Test (8 points)

A rubber cord (length l=200mm; quadratic cross section with edge length a_0 =2mm) is stretched by a tensile force linear-elasticly.

- a) Plot the curve of this tensile test in a stress-strain diagram (into the depicted coordinate system)! Show how to find Young's modulus with the help of the diagram (sketch it in!)
- b) Calculate Young's modulus for this rubber material.

$$[E = 25 \text{ N/mm}^2]$$



mean stress s_m in N/mm²

c) Poisson's ratio for rubber material is n=0.5. To which value a₁ will the edge length change while carrying out this tensile test.

$$[a_1 = 1.9 \text{ mm}]$$